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be at least 25 psig internal and 15 psig external.

- (d) Each cargo tank having a MAWP greater than 15 psig must be of circular cross-section.
 - (e) Each cargo tank having a-
- (1) MAWP greater than 15 psig must be "constructed and certified in conformance with Section VIII of the ASME Code" (IBR, see §171.7 of this subchapter); or
- (2) MAWP of 15 psig or less must be "constructed in accordance with Section VIII of the ASME Code," except as modified herein:
- (i) The recordkeeping requirements contained in Section VIII of the ASME Code do not apply. Parts UG-90 through 94 in Section VIII do not apply. Inspection and certification must be made by an inspector registered in accordance with subpart F of part 107.
- (ii) Loadings must be as prescribed in $\S 178.345-3$.
- (iii) The knuckle radius of flanged heads must be at least three times the material thickness, and in no case less than 0.5 inch. Stuffed (inserted) heads may be attached to the shell by a fillet weld. The knuckle radius and dish radius versus diameter limitations of UG-32 do not apply for cargo tank motor vehicles with a MAWP of 15 psig or less. Shell sections of cargo tanks designed with a non-circular cross sec-

tion need not be given a preliminary curvature, as prescribed in UG-79(b).

- (iv) Marking, certification, data reports, and nameplates must be as prescribed in §§ 178.345–14 and 178.345–15.
- (v) Manhole closure assemblies must conform to §§ 178.345–5.
- (vi) Pressure relief devices must be as prescribed in §178.348-4.
- (vii) The hydrostatic or pneumatic test must be as prescribed in §178.348-5.
- (viii) The following paragraphs in parts UG and UW in Section VIII of the ASME Code do not apply: UG-11, UG-12, UG-22(g), UG-32(e), UG-34, UG-35, UG-44, UG-76, UG-77, UG-80, UG-81, UG-96, UG-97, UW-13(b)(2), UW-13.1(f), and the dimensional requirements found in Figure UW-13.1.

[Amdt. 178–89, 54 FR 25031, June 12, 1989, as amended at 55 FR 37065, Sept. 7, 1990; Amdt. 178–89, 56 FR 27877, June 17, 1991; 65 FR 58632, Sept. 29, 2000; 68 FR 19285, Apr. 18, 2003; 68 fR 75756, Dec. 31, 2003]

§ 178.348-2 Material and thickness of material.

(a) The type and thickness of material for DOT 412 specification cargo tanks must conform to \$178.345–2, but in no case may the thickness be less than that determined by the minimum thickness requirements in \$178.320(a). The following Tables I and II identify the "Specified Minimum Thickness" values to be employed in that determination.

TABLE I—SPECIFIED MINIMUM THICKNESS OF HEADS (OR BULKHEADS AND BAFFLES WHEN USED AS TANK REINFORCEMENT) USING MILD STEEL (MS), HIGH STRENGTH LOW ALLOY STEEL (HSLA), AUSTENITIC STAINLESS STEEL (SS), OR ALUMINUM (AL)—EXPRESSED IN DECIMALS OF AN INCL. A FTER EXPRESSED.

INCH AFTER FORMING														
Volume capacity (gallons per inch)		10 or less	less			Over 10 to 14	to 14		ŏ	Over 14 to 18	8	18	18 and over	
Lading density at 60 °F in pounds per gallon	10 lbs and less .100	Over 10 to 13 lbs .129	Over 13 to 16 lbs .157	Over 16 lbs .187	10 lbs and less .129 .187	Over 10 to 13 lbs	Over 13 to 16 lbs270	Over 16 lbs .250 .360	10 lbs and less .157	Over 10 to 13 lbs .250	Over 13 to 16 lbs .250 .360	10 lbs and less .157	Over 10 to 13 lbs250360	Over 13 to 16 lbs312450

TABLE II—SPECIFIED MINIMUM THICKNESS OF SHELL USING MILD STEEL (MS), HIGH STRENGTH LOW ALLOY STEEL (HSLA), AUSTENITIC STAINLESS STEEL (SS), OR ALUMINUM (AL)—EXPRESSED IN DECIMALS OF AN INCH AFTER FORMING

Volume capacity in gallons per inch		10 or less	ess			Over 10 to 14	to 14		ò	Over 14 to 18	18	18	18 and over	
Lading density at 60 °F in pounds per gallon	10 lbs	Over	Over	Over	10 lbs	Over	Over	Over	10 lbs	Over	Over	10 lbs	Over	Over
	and	10 to 13 lbs	13 to 16 lbs	16 lbs	and	10 to 13 lbs	13 to 16 lbs	16 lbs	and	10 to 13 lbs	13 to 16 lbs	and	10 to 13 lbs	13 to 16 lbs
Thickness (steel):														
Distances between heads (and bulkheads baffles and ring														
stiffeners when used as tank reinforcement):														
36 in. or less	100	.129	.157	.187	100	.129	.157	.187	100	.129	.157	.129	.157	.187
Over 36 in. to 54 inches	100	.129	.157	.187	100	.129	.157	.187	.129	.157	.187	.157	.250	.250
Over 54 in. to 60 inches	.100	.129	.157	.187	.129	.157	.187	.250	.157	.250	.250	.187	.250	.312
Thickness (aluminum):														
Distances between heads (and bulkheads baffles and ring														
stiffeners when used as tank reinforcement):														
36 in. or less	144	.187	.227	.270	144	.187	.227	.270	144	.187	.227	.187	.227	.270
Over 36 in. to 54 inches	144	.187	.227	.270	144	.187	.227	.270	.187	.227	.270	.157	360	360
Over 54 in. to 60 inches	144	.187	.227	.270	.187	.227	.270	.360	.227	360	360	.270	.360	.450

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(b) [Reserved]

[Amdt. 178–89, 54 FR 25031, June 12, 1989; 54 FR 28750, July 7, 1989, as amended at 55 FR 37065, Sept. 7, 1990; 68 FR 19285, Apr. 18, 2003]

§178.348–3 Pumps, piping, hoses and connections.

Each pump and all piping, hoses and connections on each cargo tank motor vehicle must conform to §178.345-9, except that the use of nonmetallic pipes, valves, or connections are authorized on DOT 412 cargo tanks.

[Amdt. 178-89, 55 FR 37065, Sept. 7, 1990. Redesignated by Amdt. 178-112, 61 FR 18934, Apr. 29, 1996]

§178.348-4 Pressure relief.

- (a) Each cargo tank must be equipped with a pressure and vacuum relief system in accordance with §178.345–10 and this section.
- (b) Type and construction. Vacuum relief devices are not required for cargo tanks designed to be loaded by vacuum or built to withstand full vacuum.
- (c) Pressure settings of relief valves. The setting of the pressure relief devices must be in accordance with §178.345–10(d), except as provided in paragraph (d)(3) of this section.
- (d) Venting capacities. (1) The vacuum relief system must limit the vacuum to less than 80 percent of the design vacuum capability of the cargo tank.
- (2) If pressure loading or unloading devices are provided, the pressure relief system must have adequate vapor and liquid capacity to limit tank pressure to the cargo tank test pressure at the maximum loading or unloading rate. The maximum loading and unloading rates must be included on the metal specification plate.
- (3) Cargo tanks used in dedicated service for materials classed as corrosive material, with no secondary hazard, may have a total venting capacity which is less than required by \$178.345–10(e). The minimum total venting capacity for these cargo tanks must be determined in accordance with the following formula (use of approximate values given for the formula is acceptable):

Formula in Nonmetric Units

 $Q = 37,980,000 A^{0.82} (ZT)^{0.5} / (LC)(M^{0.5})$

Where:

- Q = The total required venting capacity, in cubic meters of air per hour at standard conditions of 15.6 °C and 1 atm (cubic feet of air per hour at standard conditions of 60 °F and 14.7 psia);
- T = The absolute temperature of the vapor at the venting conditions—degrees Kelvin (°C+273) [degrees Rankine (°F+460)];
- A = The exposed surface area of tank shell—square meters (square feet);
- L = The latent heat of vaporization of the lading—calories per gram (BTU/lb);
- Z = The compressibility factor for the vapor (if this factor is unknown, let Z equal 1.0):
- M = The molecular weight of vapor;
- C = A constant derived from (K), the ratio of specific heats of the vapor. If (K) is unknown, let C = 315.
- ${\rm C} = 520[{\rm K}(2/({\rm K+1}))^{[({\rm K+1})/({\rm K-1})]}]^{0.5}$

Where:

 $K = C_p / C_v$

- C_p = The specific heat at constant pressure, in -calories per gram degree centigrade (BTU/lb °F.); and
- C_v = The specific heat at constant volume, in -calories per gram degree centigrade (BTU/lb °F.).

[Amdt. 178–89, 54 FR 25032, June 12, 1989, as amended at 55 FR 37065, Sept. 7, 1990; Amdt. 178–104, 59 FR 49135, Sept. 26, 1994. Redesignated by Amdt. 178–112, 61 FR 18934, Apr. 29, 1996; 72 FR 55696, Oct. 1, 2007; 72 FR 59146, Oct. 18, 2007]

§ 178.348-5 Pressure and leakage test.

- (a) Each cargo tank must be tested in accordance with §178.345–13 and this section.
- (b) Pressure test. Test pressure must be as follows:
- (1) Using the hydrostatic test method, the test pressure must be at least 1.5 times MAWP.
- (2) Using the pneumatic test method, the test pressure must be at least 1.5 times tank MAWP, and the inspection pressure is tank MAWP.

[Amdt. 178–89, 54 FR 25032, June 12, 1989. Redesignated by Amdt. 178–112, 61 FR 18934, Apr. 29, 1996]